

fused coal ash deposits because of the lowered temperature in the gasification reactor. This causes troubles such as clog in a withdrawing line for fused ash. In order to prevent the troubles, the temperature in the furnace must be prevented from lowering. Accordingly a larger
5 amount of oxygen than a theoretical amount calculated from an elemental composition of the coal is fed to a gasification reactor in the aforesaid conventional method.

In order to use pulverized coal containing ash of a high fusing
10 temperature, especially in gasification, the inner temperature of a gasification reactor must be maintained at a relatively high temperature. Accordingly, it is difficult in the conventional methods to use coal containing ash of a high fusing temperature. When such a coal containing
15 ash of a high fusing temperature is unavoidably used, an expensive melting point depressant should be used. In addition, a larger amount of oxygen is required to somewhat raise the inner temperature of the gasification reactor whereby melting of the coal ash in the gasification reactor is promoted so as to facilitate removal of the coal ash at a
20 bottom of the gasification reactor and thereby to put the gasification plant in smooth operations. A gasification efficiency of the conventional methods is low due to these factors.

A method of coal gasification by feeding coal and water to a gasification reactor is known, where at least a part of water is fed
25 in a form of steam to the gasification reactor (see Japanese Patent Laid-open No. 2002-155288). According to the method, coal is fed with an aid of steam to a gasification reactor. Therefore, water contained in the mixture of coal and water, preferably the entire amount thereof, is vaporized into steam before being fed to the gasification reactor
30 and, therefore, the above-described drawbacks can be solved.

In the above method, a mixture of solid-liquid system is converted into a mixture of gas-solid or gas-liquid-solid system and fed to a reactor. As equipment by which a slurry of solid-liquid system is fed
35 to a heat exchanger consecutively, heated, converted into a gas-solid system or gas-liquid-solid system, which is then fed to vaporization

equipment to recover water, Cracksystem™ is commercially available from Hosokawa Micron Co., Ltd. However, the solvent vaporizes at once in the heat exchanger in this equipment, so that a flow rate of the gas-solid system at an outlet of the heat exchanger exceeds the sonic speed.
5 Accordingly, if the equipment is used for a burnable solid such as coal, heavy abrasion will take place.

In 1979, patent application was filed by the Department of Energy, United States, in which a coal water mixture, CWM, is heated and separated
10 into gas and solid in a flash dryer vessel, and then pulverized coal is fed to a gasification reactor (see United States Patent No. 4153427). However, the pulverized coal obtained in the gas-solid separation is not completely dried and, therefore, coagulates, so that a continuous feeding to the gasification reactor is difficult. Accordingly the method
15 has not been put to a practical application.

SUMMARY OF THE INVENTION

The present invention provides a method for feeding a mixture comprising a burnable solid and water to a combustion furnace or
20 gasification reactor wherein at least a part of the water in the mixture is converted into a form of steam, and wherein almost no abrasion takes place in piping and stable feeding to a combustion furnace or gasification reactor is possible without sedimentation of the burnable solid.

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In the conventional methods for feeding coal and water to a gasification reactor to gasify the coal, there was a problem that heavy abrasion takes place in pipes in the heater and a feeding pipe connected to the gasification reactor, if at least a part of the water is fed in
30 a form of steam. In order to solve this problem, it might be thought to make inner diameters of pipes in a heater and a feeding pipe large enough to reduce a flow rate of the fluid. However, if the inner diameters are made large enough to suppress abrasion, other problems, in turn, arises that coal subsides on the inner wall of the pipes and further
35 smooth conveying of the coal becomes difficult.

The inventors have made various researches to solve these problems. As a result, the inventors have found that in pumping a mixture comprising a burnable solid and water to a combustion furnace or gasification reactor if a discharge pressure is controlled so as to be in the following
5 relatively high specific range, the flow rate of the mixture may be controlled properly with diameters of the pipes being set in a proper range to thereby feed the mixture to the above-mentioned reactors in a stable manner without abrasion or sedimentation of the burnable solid in the pipes through which the mixture flows.

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Thus the present invention provides

(1) a method for feeding a mixture comprising a burnable solid and water to a combustion furnace or gasification reactor, comprising heating the mixture with a heater to convert at least a part of the water in the
15 mixture into a form of steam and feeding the whole mixture to a combustion furnace or gasification reactor, wherein the whole mixture is transferred between the heater and the combustion furnace or gasification reactor by a pump, characterized in that a discharge pressure at the pump is higher than an the inner pressure in the combustion
20 furnace or gasification reactor at least by 1.5 MPa and not higher than 22.12 MPa, and that a flow rate of said mixture with at least a part of the water in the mixture being in a form of steam is from 6 to 50 m/s in a pipe within the heater and in a pipe between an outlet of the heater and an inlet of the combustion furnace or gasification reactor.

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Preferred embodiments are as follows:

(2) the method according to the above-described (1), wherein a discharge pressure at the pump is higher than the inner pressure of the combustion furnace or gasification reactor by from 3.0 MPa to 15.0 MPa;
30 (3) the method according to the above-described (1), wherein a discharge pressure at the pump is higher than an inner pressure in the combustion furnace or gasification reactor by from 4.0 MPa to 15.0 MPa;
(4) the method according to any one of the above-described (1)-(3), wherein the above-described flow rate is from 8 to 40 m/s;
35 (5) the method according to any one of the above-described (1)-(3), wherein the above-described flow rate is from 10 to 40 m/s;

- (6) the method according to any one of the above-described (1)-(5), wherein an inner diameter of the pipe in the heater becomes larger gradually along a direction of the flow of the mixture, so that the water is gradually converted into a form of steam;
- 5 (7) the method according to any one of the above-described (1)-(5), wherein an inner diameter of the pipe in the heater becomes larger stepwise along a direction of the flow of the mixture, so that the water is converted stepwise into a form of steam;
- (8) the method according to any one of the above-described (7), wherein
- 10 a pressure reducing valve is provided between sections of the pipe with different diameters, so that the water in the mixture is converted into a form of steam with an aid of the pressure reducing valve;
- (9) the method according to any one of the above-described (7) or (8), wherein an inner diameter of the pipe in the heater becomes larger in
- 15 from two to twelve steps;
- (10) the method according to any one of the above-described (7) or (8), wherein an inner diameter of the pipe in the heater becomes larger in from four to twelve steps;
- (11) the method according to any one of the above-described (7) or (8),
- 20 wherein an inner diameter of the pipe in the heater becomes larger in from six to twelve steps;
- (12) the method according to any one of the above-described (7)-(11), wherein a non-flammable gas is blown in just downstream of a place where an inner diameter of the pipe becomes larger or a place where a pressure
- 25 reducing valve is provided;
- (13) the method according to any one of the above-described (12), wherein the non-flammable gas is steam, nitrogen, or gaseous carbon dioxide;
- (14) the method according to any one of the above-described (1)-(13), wherein substantially all of the water is converted into a form of steam;
- 30 (15) the method according to any one of the above-described (1)-(14), wherein heating by the heater is carried out at a temperature of from 150 to 450 degrees C at a pressure of from 1.5 to 22.12 MPa;
- (16) the method according to any one of the above-described (1)-(14), wherein heating by the heater is carried out at a temperature of from
- 35 200 to 400 degrees C at a pressure of from 3.0 to 22.12 MPa;
- (17) the method according to any one of the above-described (1)-(14),

wherein heating by the heater is carried out at a temperature of from 200 to 365 degrees C at a pressure of from 4.0 to 20.0 MPa;

(18) the method according to any one of the above-described (1)-(17), wherein heating is carried out with a heating medium at a temperature
5 of from 200 to 600 degrees C;

(19) the method according to any one of the above-described (1)-(18), wherein a pressure control valve is provided between the outlet of the heater and the combustion furnace or gasification reactor;

(20) the method according to any one of the above-described (1)-(19),
10 wherein a pre-heater is provided upstream of the heater;

(21) the method according to the above-described (20), wherein a pressure reducing valve is provided at the outlet of the pre-heater;

(22) the method according to any one of the above-described (1)-(21), wherein a water content in the mixture comprising a burnable solid and
15 water is from 27 to 80 weight %, relative to the total weight of the mixture;

(23) the method according to any one of the above-described (1)-(21), wherein a water content in the mixture comprising a burnable solid and
20 water is from 30 to 40 weight %, relative to the total weight of the mixture; and

(24) the method according to any one of the above-described (1)-(21), wherein a water content in the mixture comprising a burnable solid and
water is from 30 to 35 weight %, relative to the total weight of the mixture.

BRIEF DESCRIPTION OF THE DRAWINGS

Figure 1 is a process flow chart of the device used in the Examples.
Figure 2 is a graph to show a profile of the flow rate in the pipe between
the outlet of the pump and the inlet of the gasification reactor in the
30 Example 1.

Figure 3 is a graph to show a profile of the pressure in the pipe between
the outlet of the pump and the inlet of the gasification reactor in Example
1.

Figure 4 is a graph to show a profile of the flow rate in the pipe between
35 the outlet of the pump and the inlet of the gasification reactor in Example
2.

Figure 5 is a graph to show a profile of the flow rate in the pipe between the outlet of the pump and the inlet of the gasification reactor in Example 2.

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DESCRIPTION OF THE PREFERRED EMBODIMENTS

For the water content of the mixture comprising a burnable solid and water used in the present invention, the upper limit is preferably 80 weight %, more preferably 40 weight %, and even more preferably 35 weight %, and the lower limit is preferably 27 weight %, and more preferably 30 weight %. Meanwhile, for the content of the burnable solid, the upper limit is preferably 73 weight %, and more preferably 70 weight %, and the lower limit is preferably 20 weight %, more preferably 60 weight %, and even more preferably 65 weight %. If the water content exceeds the aforementioned upper limit and the content of the burnable solid is less than the aforementioned lower limit, the energy to vaporize water is too large for the present method to be economical. If the water content is less than the aforementioned lower limit and the content of the burnable solid exceeds the aforementioned upper limit, the mixture comprising the burnable solid and water is too viscous to be transferred smoothly. A surfactant may be added to facilitate formation of an aqueous slurry of a burnable solid.

The burnable solids subjected to combustion or gasification are not particularly limited to specific kinds. Use may be made of, for instance, coal, coal or petroleum coke, coal or petroleum pitch, or cellulosic solid waste. Coal of various coal ranks may be used, such as bituminous coal, sub-bituminous coal, or brown coal. Coal containing an ash of a high melting point is difficult to use in a conventional method where a coal/water slurry is fed to a gasification reactor. In the present invention, such a limitation caused by melting point of ash is imposed. The burnable solid is preferably pulverized to a desired grain size before used. The grain size is preferably from 25 to 500 meshes, more preferably from 50 to 200 meshes. If the grain size of the coal is too large, the coal particles cause very fast sedimentation in water. The coal is pulverized preferably in a dry state before mixed with water, but it may be also pulverized in a wet state after mixed with water.

The mixture comprising a burnable solid and water is fed by a pump to a combustion furnace or gasification furnace through a heater. As the pump, any known pump may be used, and mention may be made of, for instance, a centrifugal pump, a plunger pump, or a gear pump.

The upper limit of the discharge pressure of the pump in the present invention is 22.12MPa, which is the saturated steam pressure at the critical temperature of water, 374.15 degrees C. The pressure is preferably higher than a pressure in the combustion furnace or gasification furnace by 15.0 MPa, and more preferably by 10.0 MPa. The lower limit is a pressure higher than a pressure in the combustion furnace or gasification furnace by 1.5 MPa, preferably by 3.0 MPa, and more preferably by 4.0 MPa. If the pressure exceeds the aforesaid upper limit, large costs are needed to make apparatuses pressure-proof and, therefore, the method is uneconomical. If the pressure is lower than the aforementioned lower limit, more water will vaporize than desired and, therefore, the flow rate of the mixture becomes lower than the required flow rate as described below and, therefore, the burnable mixture sometimes cannot be smoothly transferred to the gasification reactor.

As the heater in the present invention, use may be made of any heater that can heat the above-described mixture and convert at least a part of the water in the mixture, preferably substantially all of the water, into a form of steam. For instance, a heating furnace or a heat exchanger may be used. Preferably, a heat exchanger, more preferably a double tube heat exchanger, may be used.

In the present invention, it is needed that the flow rate of the aforementioned mixture in the pipe of the heater and in the pipe between the outlet of the heater and the inlet of the combustion furnace or gasification reactor be in the following range; the upper limit of the flow rate: 50 m/s, preferably 40 m/s, and more preferably 30 m/s; and the lower limit: 6 m/s, preferably 8 m/s, and more preferably 10 m/s. Thereby the mixture may be fed to the combustion furnace or gasification reactor in a stable manner. If the flow rate exceeds the aforementioned

upper limit, the pipes wear out heavily. If the flow rate is lower than the aforementioned lower limit, the pipes easily clog because of the sedimentation of the burnable solid.

5 The inner diameter of the pipe in the heater through which the mixture comprising a burnable solid and water passes preferably becomes larger gradually, and more preferably stepwise along a direction of the flow of the mixture. Thereby the water in the mixture may be converted into a form of steam gradually or stepwise to control the flow rate of
10 the mixture properly. In an embodiment wherein an inner diameter of the pipe becomes larger stepwise, the inner diameter becomes larger in from 2 to 12 steps, more preferably in from 4 to 12 steps, even more preferably in from 6 to 12 steps. It is also preferred that pressure reducing valves are provided between sections of the pipe with different inner diameters,
15 whereby a desired amount of the water in the mixture may be converted into a form of steam properly. Preferably a non-flammable gas is blown just downstream of a place where an inner diameter of a pipe becomes larger or just downstream of a place where a pressure reducing valve is provided. As the non-flammable gas, steam, nitrogen, or carbon dioxide
20 is preferably used. By blowing the non-flammable gas in, it is possible to prevent the flow rate of the mixture in the pipe from lowering, and to thereby maintain the flow rate of the mixture in the pipes in the desired range described above.

25 In the heater, the aforementioned mixture is heated to a temperature at which at least a part, preferably substantially all, of the water in the mixture vaporizes and is converted into a form of steam. The upper limit of the heating temperature is preferably 450 degrees C, more preferably 400 degrees C, and particularly preferably 365 degrees
30 C. The lower limit is preferably 150 degrees C, more preferably 200 degrees C, and even more preferably, 250 degrees C. If the temperature exceeds the aforementioned upper limit, a burnable solid, such as coal, causes an intense thermal decomposition and the resulting hydrocarbon substances often cause coking in the pipes, which leads to choke of the
35 pipes in the heater. Below the lower limit, water may not be sufficiently vaporized. The pressure in the pipe in the heater during the heating

described above depends on a discharge pressure of the pump and is preferably from 1.5 to 22.12 MPa, more preferably from 3.0 to 22.12 MPa, and even more preferably from 4.0 to 20.0 MPa.

- 5 The aforementioned heating is carried out preferably by a heating medium, preferably heating oil or fused salt in a heat exchanger, such as a double tube heat exchanger. A temperature of the heating medium is preferably from 200 to 600 degrees C, more preferably from 250 to 500 degrees C, and particularly preferably from 300 to 450 degrees C.
- 10 If the temperature exceeds the aforementioned upper limit, a burnable solid, such as coal, causes thermal decomposition and the resulting hydrocarbon substances cause coking, which often leads to the choke of the pipe in the heater. Below the aforementioned lower limit, it is difficult to heat the mixture to the desired temperature described above.
- 15 A heater for heating the heating medium is not particularly restricted and any heater that can heat the heating medium to the desired temperature described above may be used. Preferably a heat exchanger using a heating medium such as hot steam, hot oils, fused salts or gases may be used.
- 20 In the present invention, a pre-heater may be provided to heat the mixture before the mixture is heated in the above-described heater, whereby the temperature at which the mixture is fed to a combustion furnace or gasification reactor may be controlled properly, depending upon an operation temperature of the combustion furnace or gasification
- 25 reactor. The upper limit of the pre-heating temperature is preferably 450 degrees C, more preferably 400 degrees C, and even more preferably 365 degrees C. The lower limit is preferably 150 degrees C, more preferably 200 degrees C, and even more preferably 250 degrees C. The pressure in the pre-heating may be similar with the discharge pressure
- 30 of the pump. The pre-heater aims to heat the mixture to a certain temperature and, therefore, the pressure in a pipe in the pre-heater is preferably equal to or higher than the saturated vapor pressure at the aforementioned desired temperature so as to prevent the water in the mixture from evaporating. In order to keep this pressure, a pressure
- 35 control valve may preferably be provided at an outlet of the pre-heater.

The mixture comprising a burnable solid and water is heated to the aforementioned desired temperature in the heater, and at least a part, more preferably substantially all, preferably 95 weight % or more, and more preferably 98 weight % or more, of the water is converted into steam. The resulting steam pneumatically conveys the burnable solid and feeds it to the combustion furnace or gasification reactor. The combustion furnace is preferably kept at a temperature of from 1,300 to 2,000 degrees C, and more preferably from 1,300 to 1,700 degrees C, under an atmospheric pressure or slightly pressurized condition to burn the introduced burnable solid. Meanwhile, the gasification reactor is kept at a temperature of from 1,000 to 2,500 degrees C, more preferably from 1,300 to 2,000 degrees C at a pressure of from 0.5 to 10 MPa, more preferably from 1 to 10 MPa, and even more preferably 2 to 10 MPa, to gasify the introduced burnable solid. The combustion furnace or gasification reactor is preferably provided with a pressure control valve, capable of being fully closed, at the inlet, so that the amount of the mixture to be fed to the furnace may be properly controlled.

The method of the present invention may be applied to any known combustion or gasification methods to burn or gasify a mixture containing a burnable solid and water. As the gasification method, Texaco method and the Dow method may be mentioned.

The present invention will be explained in detail with reference to the following Examples, but shall not be limited thereto.

EXAMPLES

Example 1

The process flow shown in FIG. 1 was used in Example 1, wherein 1 is a tank; 2, a pump; 3, a pipe; 4, a heater for a heating medium; 5, a pre-heater; 6, a pressure control valve; 7, a first heater; 8, a second heater; 9, a third heater; 10, a fourth heater; 11, a pipe; 12, a pressure control valve; and 13, a gasification reactor. As the burnable solid, pulverized coal A, general coal with a grain size of from 50 to 200 meshes, was used. The pulverized coal was mixed with a given amount of water in a slurry maker, not shown, to prepare a mixture of coal and

water. The mixture was placed in tank 1 and stirring was continued to prevent sedimentation of the pulverized coal. The coal and water contents and the viscosity of the mixture and the Higher Heating Value, the ash content of the coal, and the melting point of the ash from the coal are as shown in the following Table 1.

Table 1

Mixture

Coal Content	50.0 weight %
Water Content	50.0 weight %
Viscosity	4000 cp at 20 deg. C to 170 cp at 95 deg. C

Coal

Ash Content	4.3 weight %
Higher Heating Value (HHV)	3210 kcal/kg
Melting Point of Ash	1150 deg. C

10 The above-described mixture of coal and water was pressurized with
an aid of pump 2 to 11.76 MPa (120 kg/cm²), and then conveyed to pre-heater
5 through line 3 in a flow amount of 130 kg/hour. The mixture pipe in
pre-heater 5 was 6mm in inner diameter and 80 m in total length. Here,
the mixture was pre-heated to 300 degrees C with a heating medium of
15 340 degrees C in a heater for heating medium 4. In order to prevent the
water in the mixture from evaporating in pre-heater 5 and also to
compensate pressure loss, the pressure in the pump side of the mixture
pipe was maintained at a pressure of 10.58 MPa, (108 kg/cm²), which is
higher than the saturated vapor pressure of water at 300 degrees C,
20 approximately 8.82 MPa (approximately 90 kg/cm²). The flow rate of the
mixture in the pipe of pre-heater 5 was 1.16 m/s.

 The mixture pre-heated to 300 degrees C in pre-heater 5 was
transferred to first heater 7 via pressure control valve 6. The mixture
25 pipe of first heater 7 was composed of a pipe of 2 mm in inner diameter
x 2 m long, a pipe of 3 mm in inner diameter x 4 m long, and a pipe of
4 mm in inner diameter x 4 m long, the total length, 10 m along a direction

of the flow of the mixture. The mixture was heated also in this pipe with a heating medium of 340 degrees C. In first heater 7, a part of the water of the mixture evaporated. The flow rate of the mixture in the pipe of first heater 7 was 11.5 m/s at a pressure of 9.18 MPa (93.7 kg/cm²) at the inlet, with the inner diameter being 2 mm, and 27.95 m/s at the outlet, with the inner diameter being 4 mm. The temperature at the outlet was 268 degrees C and the pressure at the outlet was 5.24 MPa (53.5 kg/cm²).

10 The mixture which left first heater 7 was conveyed to second heater 8. A mixture pipe in second heater 8 was 6 mm in inner diameter and 10 m in total length. Here the mixture was again heated with a heating medium of 340 degrees C. Further, a part of the water in the mixture vaporized due to the adiabatic expansion in second heater 8. The flow rate of the mixture in the pipe of second heater 8 was 12.55 m/s at the inlet and 29.25 m/s at the outlet. At the outlet the temperature was 255 degrees C and the pressure was 4.19 MPa (42.8 kg/cm²).

20 The mixture which left second heater 8 was then conveyed to third heater 9. A mixture pipe in third heater 9 was 8 mm in inner diameter and 10 m in total length. Here the mixture was again heated with a heating medium of 340 degrees C. Further, a part of the water in the mixture vaporized due to the adiabatic expansion in third heater 9. The flow rate of the mixture in the pipe of third heater 9 was 16.45 m/s at the inlet and 33.02 m/s at the outlet. At the outlet the temperature was 245 degrees C and the pressure was 2.8 MPa (28.6 kg/cm²).

30 The mixture which left third heater 9 was conveyed to fourth heater 10. The mixture pipe in fourth heater 10 was 12mm in inner diameter and 30 m in total length. Here the mixture was again heated with a heating medium of 340 degrees C. Further, a part of the water in the mixture vaporized due to the adiabatic expansion in fourth heater 10 and, after all, substantially all of the water in the mixture introduced into the heaters was converted into steam. The flow rate of the mixture in the pipe of fourth heater 10 was 11.3 m/s at the inlet and 35.76 m/s at the outlet. At the outlet the temperature was 300 degrees C and the pressure

was 1.96 MPa (20 kg/cm²).

The mixture thus heated was introduced via line 11 and control valve 12 to gasification reactor 13 where the pressure was maintained at 1.96 MPa (20 kg/cm²). In the gasification reactor, the pulverized coal was gasified according to a known method. The flow rate in line 11 was almost equal to that at the outlet of fourth heater 10.

Figs. 2 and 3 show changes in flow rates and pressures between the outlet of pump 2 and gasification reactor 13. The flow rate of the mixture was calculated from the pressures and temperatures in the pipes of each heater.

The above steps were continued for 50 hours, during which stable operations could be attained without sedimentation of the pulverized coal. After the operations, the pipe connecting to the gasification reactor and the inlet and the outlet of the control valve 12, where the flow rate through the piping became fastest, were inspected visually to find almost no abrasion on each inner wall.

Example 2

In Example 2, the same process flow as in Example 1, shown in Fig.1, was used. The viscosity of the mixture used in Example 2 was different from that of the mixture used in Example 1 since the type of the pulverized coal was different as shown below. Accordingly, lengths of the mixture pipes in the pre-heater and the heaters were changed so that stable operations would be secured over a long time. As the burnable solid, pulverized coal B, general coal with a grain size of from 50 to 200 meshes, was used instead of the pulverized coal A to prepare a mixture of coal and water according to the same procedures as Example 1. The coal and water contents and the viscosity of the mixture and the higher heating value, the ash content, and the melting point of the ash of the coal are as shown in the following Table. 2.

Table 2

Mixture

Coal Content	50.0 weight %
Water Content	50.0 weight %
Viscosity	400 cp at 20 deg. C to 70 cp at 95 deg. C

Coal

Ash Content	9.5 weight %
Higher Heating Value (HHV)	7090 kcal/kg
Melting Point of the Ash	1450 deg. C

A mixture of the aforesaid coal and water was pressurized to 9.87 MPa (100.6 kg/cm²) with pump 2 and then was conveyed to pre-heater 5 via line 3 in a flow amount of 140 kg/hour. A mixture pipe in pre-heater 5 was 6 mm in inner diameter and 73 m in total length. In this pipe, the mixture was pre-heated to 300 degrees C with a heating medium heated to 310 degrees C in a heater for heating medium 4. In order to prevent water in the mixture from vaporizing in pre-heater 5 and to compensate pressure loss, the pressure in the pump side of the mixture pipe was maintained at 9.25 MPa (94.3 kg/cm²), which pressure was higher than the saturated vapor pressure of water at 300 degrees C, approximately 8.82 MPa (approximately 90 kg/cm²). The flow rate of the mixture in the pipe of pre-heater 5 was 1.3 m/s.

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The mixture pre-heated to 300 degrees C in pre-heater 5 was conveyed to first heater 7 via pressure control valve 6. The mixture pipe of first heater 7 was composed of a pipe of 2 mm in inner diameter x 3 m long, a pipe of 3 mm in inner diameter x 2 m long, and a pipe of 4 mm in inner diameter x 2 m long joined in this order toward the gasification reactor along with the direction of the flow, and the total length was 7 m. The mixture was again heated in this pipe with a heating medium of 310 degrees C. In first heater 7, a part of the water of the mixture vaporized. The flow rate of the mixture in first heater 7 was 13.4 m/s at a pressure of 8.97 MPa (91.5 kg/cm²) at the inlet of the pipe, with the inner diameter of 2 mm, and 23.7 m/s at the outlet of the pipe, with the inner diameter of 4 mm. The temperature at the outlet was 252 degrees C and the pressure at the outlet was 4.03 MPa (41.1 kg/cm²).

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The mixture which left first heater 7 was conveyed to second heater 8. The mixture pipe of second heater 8 was 6 mm in inner diameter and 11.5 m in total length. The mixture was heated also here with a heating medium of 310 degrees C. A part of the water in the mixture vaporized further due to the adiabatic expansion in second heater 8. The flow rate of the mixture in second heater 8 was 10.8 m/s at the inlet of the pipe and 19.9 m/s at the outlet. At the outlet the temperature was 245 degrees C and the pressure was 3.55 MPa (36.2 kg/cm²).

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The mixture which left second heater 8 was conveyed to third heater 9. The mixture pipe of third heater 9 was 8 mm in inner diameter and 16.5 m in total length. The mixture was heated also here with a heating medium of 310 degrees C. A part of the water in the mixture vaporized further due to the adiabatic expansion in third heater 9. The flow rate of the mixture in third heater 9 was 11.4 m/s at the inlet and 25.8 m/s at the outlet. At the outlet the temperature was 227 degrees C and the pressure was 2.54 MPa (25.9 kg/cm²).

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The mixture which left second heater 9 was conveyed to fourth heater 10. The mixture pipe of fourth heater 10 was 12 mm in inner diameter and 19 m in total length. The mixture was heated also here with a heating medium of 310 degrees C. A part of the water in the mixture vaporized further due to the adiabatic expansion in fourth heater 10 and, after all, substantially all of the water in the mixture introduced into the heaters was converted into steam. The flow rate of the mixture in fourth heater 10 was 11.7 m/s at the inlet and 19.9 m/s at the outlet. At the outlet the temperature was 244 degrees C and the pressure was 1.96 MPa (20 kg/cm²).

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The mixture thus heated was introduced via line 11 and control valve 12 to gasification reactor 13 maintained at a pressure of 1.96 MPa (20 kg/cm²). In the gasification reactor, the pulverized coal was gasified according to a known method. The flow rate of the mixture in line 11 was almost equal to that at the outlet of fourth heater 10.

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Profiles of the flow rates and pressures of the mixture from the outlet of pump 2 to gasification reactor 13 described above are shown in Figs. 4 and 5. The flow rate of the mixture was calculated from the pressures and temperatures in the pipes of each heater.

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The aforementioned steps were continued for 50 hours, during which stable operations were attained without sedimentation of the pulverized coal. After the operations, the pipe connecting to the gasification reactor and the inlet and the outlet of the control valve 12, where the flow rate through the pipes became fastest, were inspected visually to find almost no abrasion on each inner wall as in Example 1.

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INDUSTRIAL APPLICABILITY

The present invention provides a method for feeding a mixture comprising a burnable solid and water to a combustion furnace or gasification reactor wherein at least a part of the water in the mixture is converted into a form of steam, and wherein almost no abrasion takes place in piping and stable feeding to a combustion furnace or gasification reactor is possible without sedimentation of the burnable solid.

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